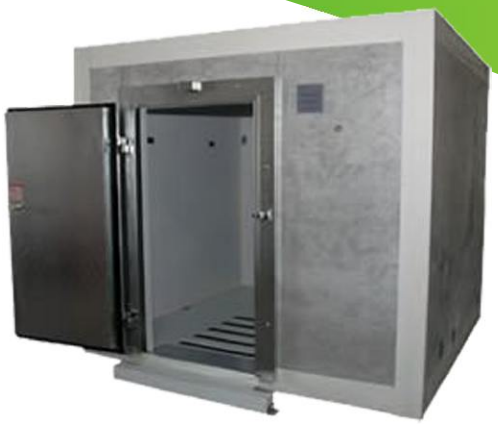
FOUR SEASONS HOTEL PHILADELPHIA



Walk-In Freezer

September 30, 2010

Mr. Marvin Dixon
Director of Engineering
Four Seasons Hotel, Philadelphia
One Logan Square
Philadelphia, PA 19103

Dear Marvin,

Thank you for the opportunity to conduct a Proof of Performance with the installation of IceCOLD® into a walk-in freezer located in the main kitchen at The Four Seasons Hotel, Philadelphia.

Using a defined protocol methodology, we conducted the Proof of Performance on this installation to illustrate the savings the Four Seasons Hotel will garner as a result of introducing IceCOLD® in their refrigeration units.

The following report outlines the methodology, results of the Proof of Performance, efficiency improvement and ultimate savings obtained:

METHODOLOGY

Purpose:
Establish a baseline for electricity cost, return and supply air temperatures, under normal operating conditions of a walk-in unit at Four Seasons Hotel in Philadelphia. Measure the difference between cost of operating unit before IceCOLD® and after IceCOLD® to determine efficiency improvement and expected energy savings.

Brands: Heatcraft/Bohn & Copeland
Use: Refrigeration
Refrigerant: R22
IceCOLD® Installed: 2 ounces

Tools Utilized:
U12 Data Logger (Onset Computer)
2 Temperature / Relative Humidity Sensors
1 AMP Probes

Probe Placement:
Channel 1: AMP Probe (lower amp reading) placed on the line side voltage on the evaporator
Channel 2: Temp/RH probe (lower temp reading) placed on the supply air from the evaporator)
Channel 3: Temp/RH probe (higher temp reading) placed on the return air on the evaporator

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Units Measured:
Amperage: Current draw on the L1 (overall amp draw) compressor unit
Temperature/Relative Humidity: On the return and supply air on the evaporator unit

Volume of IceCOLD® Installed:
Using a standard formula ratio of 1 ounce of IceCOLD® to 1 ton, 2 ounces were installed on Wednesday, September 1, 2010.

Units Measured:
AMP: Amount of electricity needed to operate system
Temperature/Relative Humidity: affecting performance of system

The data logger collected the following data:
As this is a refrigeration unit (walk-in freezer), no cooling degree days were used in the calculations. Improvement is based on reduced electricity consumption and improved cold air into the unit. To ensure a comparative analysis, four days were chosen in the baseline period for comparison to the exact same days in the post period. Three of those days were Friday, Saturday and Sunday which are typically very busy in a hotel environment with one weekday (Tuesday) to provide a weekday analysis.

Baseline: July 20, July 30, July 24, July 25, 2010
Average amps = 4.279 + 2.25 + 3.08 + 2.08 = 2.9
Return Air average = 42F
Supply Air Average = 26.99 F
Delta = 15.1F

Post Installation: September 10, 11, 12, 14, 2010
Average amps = 2.57421 + 1.991997 + 1.99904 + 1.764914 = 2.075
Return Air Average = 36.86 + 36.73 + 36.83+ 38.83 = 37.31
Supply Air Average 33.68 + 33.57 + 33.82 + 34.59 = 33.89
Delta = 3.4 F

There was a decrease in amps from an average of 2.9 Amps to 2.075 Amps
Reduction of 28.44% Electricity Use
The post install difference in return and supply air temperature across the coils minus the baseline difference in return and supply air temperature:

15.1F – 3.4F is -11.07 degrees cooler
Overall efficiency gain is therefore: **28.44% Improvement**

PERFORMANCE IMPROVEMENT

Unit used 28% LESS power to keep the walk-in freezer cooler by reducing the return and supply air by 11.7 degrees. By reducing the delta between return and supply air the content inside the walk-in freezer is less exposed to extreme temperature changes. The compressor does not need to run as often or as long in order to meet and hold thermostat set point.

28.4% Less electricity was required for cooling of the space post installation of IceCOLD®